

# ASME BPVC VIII-1 2021

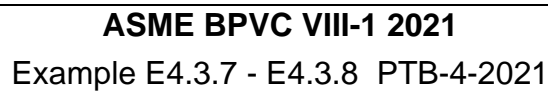
## Example E4.3.7 - E4.3.8 PTB-4-2021

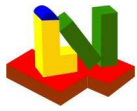
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### Layout

Input values:	1.234	or	1.234
Calculated values:	1.234	or	1.234
Critical values:	1.234	or	1.234
Estimated values:	1.234	or	1.234

[illegible]



# ASME BPVC VIII-1 2021

## Example E4.3.7 - E4.3.8 PTB-4-2021

### E4.3.7 Large End - Cone-to-cylinder junction at large end under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2021

#### Cone-to-cylinder junction (wide end)

Type of reinforcing ring

(0 = no, 1 = at the cylinder, 2 = at the cone)

Without stiffener

Design pressure

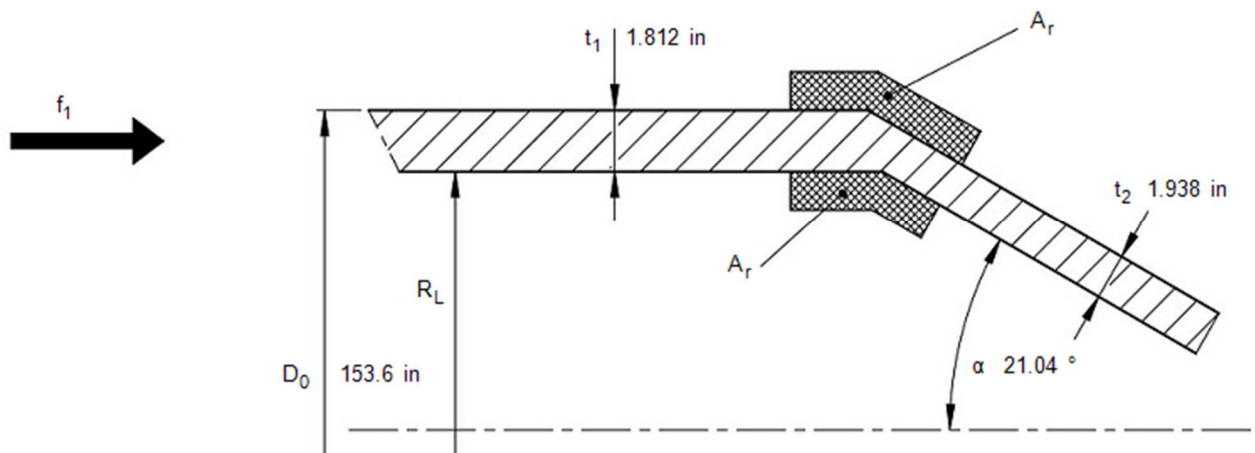
Hydrostatic head

Calculation pressure

Calculation temperature

Axial load based on circumference (for compression negative)

$p_D$	356 psi
$D_p$	0 psi
$p_0$	356 psi
$T_0$	300 °F
$f_1$	0 lbf/in



#### Cylinder

Outside diameter

Final wall thickness

Material K02700-SA-516-70-Class:-Size:

Wall thickness allowance

Allowance (corrosion)

Thickness without allowances

Inside radius

Allowable stress

Modulus of elasticity

Weld joint efficiency (or Cast Quality Factor)

(=  $D_0/2 - t_s$ )

$D_0$	153.6 in
$t_1$	1.812 in
$c_1$	0 in
$c_2$	0.125 in
$t_s$	1.687 in
$R_L$	75.12 in
$S_s$	20015 psi
$E_s$	2.829e+7 psi
$E_1$	1

#### Cone

Half-apex angle ( $\leq 30^\circ$ )

Final wall thickness

Material K02700-SA-516-70-Class:-Size:

Wall thickness allowance

Allowance (corrosion)

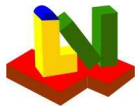
Effective thickness

Allowable stress

Modulus of elasticity

Weld joint efficiency (or Cast Quality Factor)

$\alpha$	21.04 °
$t_2$	1.938 in
$c_1$	0 in
$c_2$	0.125 in
$t_c$	1.813 in
$S_c$	20015 psi
$E_c$	2.829e+7 psi
$E_2$	1



# ASME BPVC VIII-1 2021

## Example E4.3.7 - E4.3.8 PTB-4-2021

### Results

Geometrical conditions  
**valid**

Strength condition  
**Wall thickness acceptable**

Factor			k	1
Ratio		$P_0/S_s E_1$		0.01779
Angle	(Reinforcement required if $\Delta < \alpha$ )	21.04°	$\Delta$	43.56°
Effective load			$Q_L$	13371 lbf/in
Required thickness cylinder (UG-27)			t	1.351 in
with allowances ( $t_1$ )	1.812 in	$\geq t_+$	$t_+$	1.476 in
Required thickness cone (UG-32)			$t_r$	1.447 in
with allowances ( $t_2$ )	1.938 in	$\geq t_r$	$t_r$	1.572 in
Required cross sectional area			$A_{rL}$	0 in <sup>2</sup>
Available cross section			$A_{eL}$	8.209 in <sup>2</sup>
Required area of reinforcement			$A_r$	0 in <sup>2</sup>
Available area of reinforcement		$b_v \cdot t_v$	$A_v$	0 in <sup>2</sup>
Maximum distance from the connection point of the complete reinforcing area		$\sqrt{[R_L \cdot t_s]}$		11.26 in
centroid of reinforcing area		$0.25 \cdot \sqrt{[R_L \cdot t_s]}$		2.815 in

### Equations

$$\cos(\alpha) = \cos(\alpha) \Leftrightarrow \cos(21.04^\circ) = 0.9333$$

$$\tan(\alpha) = \tan(\alpha) \Leftrightarrow \tan(21.04^\circ) = 0.3846$$

$$Q_L = P_0 \cdot \frac{R_L}{2} + f_1 = 24.55 \text{ bar} \cdot \frac{1908 \text{ mm}}{2} + 0 \text{ N/mm} = 2342 \text{ N/mm}$$

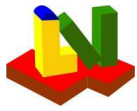
$$A_{rL} = \frac{k \cdot Q_L \cdot R_L}{S_s \cdot E_1} \cdot \left[ 1 - \frac{D_{el}}{\alpha} \right] \cdot \tan(\alpha) = \frac{1 \cdot 2342 \text{ N/mm} \cdot 1908 \text{ mm}}{138 \text{ N/mm}^2 \cdot 1} \cdot \left[ 1 - \frac{43.56^\circ}{21.04^\circ} \right] \cdot 0.3846 = 0 \text{ mm}^2$$

$$t = \frac{P_0 \cdot R_L}{S_s \cdot E_1 - 0.6 \cdot P_0} = \frac{24.55 \text{ bar} \cdot 1908 \text{ mm}}{138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 24.55 \text{ bar}} = 34.3 \text{ mm}$$

$$t_r = \frac{P_0 \cdot \frac{R_L}{\cos(\alpha)}}{S_c \cdot E_2 - 0.6 \cdot P_0} = \frac{24.55 \text{ bar} \cdot \frac{1908 \text{ mm}}{0.9333}}{138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 24.55 \text{ bar}} = 36.75 \text{ mm}$$

$$A_{eL} = (t_s - t) \cdot \sqrt{R_L \cdot t_s} + (t_c - t_r) \cdot \sqrt{R_L \cdot \frac{t_c}{\cos(\alpha)}} =$$

$$(42.86 \text{ mm} - 34.3 \text{ mm}) \cdot \sqrt{1908 \text{ mm} \cdot 42.86 \text{ mm}} + (46.04 \text{ mm} - 36.75 \text{ mm}) \cdot \sqrt{1908 \text{ mm} \cdot \frac{46.04 \text{ mm}}{0.9333}} = 5296 \text{ mm}^2$$



# ASME BPVC VIII-1 2021

## Example E4.3.7 - E4.3.8 PTB-4-2021

### E4.3.7 Small End - Cone-to-cylinder junction at small end under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2021

#### Cone-to-cylinder junction (small end)

Type of reinforcing ring

(0 = no, 1 = at the cylinder, 2 = at the cone)

0

**Without stiffener**

Design pressure

$p_D$  356 psi

Hydrostatic head

$D_p$  0 psi

Calculation pressure

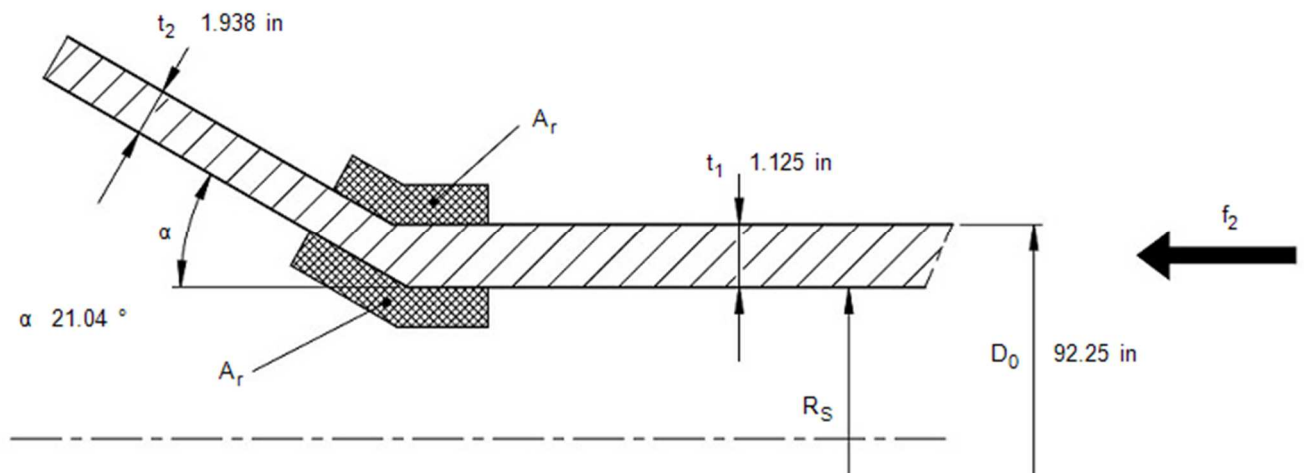
$p_0$  **356** psi

Calculation temperature

$T_0$  300 °F

Axial load based on circumference (for compression negative)

$f_2$  396.9 lbf/in



#### Cylinder

Outside diameter

$D_0$  92.25 in

Final wall thickness

$t_1$  1.125 in

Material K02700-SA-516-70-Class:-Size:

Wall thickness allowance

$c_1$  0 in

Allowance (corrosion)

$c_2$  **0.125** in

Effective thickness

$t_s$  **1** in

Inside radius

(=  $D_0/2 - t_s$ )

$R_S$  **45.12** in

Allowable stress

$S_s$  20015 psi

Modulus of elasticity

$E_s$  2.829e+7 psi

Weld joint efficiency (or Cast Quality Factor)

$E_1$  1

#### Cone

Half-apex angle ( $\leq 30^\circ$ )

$\alpha$  21.04 °

Final wall thickness

$t_2$  1.938 in

Material K02700-SA-516-70-Class:-Size:

Wall thickness allowance

$c_1$  0 in

Allowance (corrosion)

$c_2$  **0.125** in

Thickness without allowances

$t_c$  **1.813** in

Allowable stress

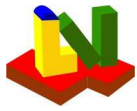
$S_c$  20015 psi

Modulus of elasticity

$E_c$  2.829e+7 psi

Weld joint efficiency (or Cast Quality Factor)

$E_2$  1



# ASME BPVC VIII-1 2021

## Example E4.3.7 - E4.3.8 PTB-4-2021

### Results

Geometrical conditions  
**valid**

Strength condition  
**Wall thickness acceptable**

Factor		k	1
Ratio	$P_0/S_s E_1$	$\Delta$	0.01779
Angle		$Q_S$	11.87 °
Effective load			8429 lbf/in
Required thickness cylinder (UG-27)		t	0.8113 in
with allowances ( $t_1$ )	1.125 in	$t_+$	0.9363 in
Required thickness cone (UG-32)		$t_r$	0.8692 in
with allowances ( $t_2$ )	1.938 in	$t_{r+}$	0.9942 in
Required cross sectional area		$A_{rS}$	3.185 in <sup>2</sup>
Available cross section		$A_{eS}$	7.876 in <sup>2</sup>
Required area of reinforcement		$A_r$	0 in <sup>2</sup>
Available area of reinforcement	$b_v \cdot t_v$	$A_v$	0 in <sup>2</sup>
Maximum distance from the connection point of the complete reinforcing area	$\sqrt{[R_L \cdot t_s]}$		9.044 in
centroid of reinforcing area	$0.25 \cdot \sqrt{[R_L \cdot t_s]}$		2.261 in

### Equations

$$\cos(\alpha) = \cos(\alpha) \Leftrightarrow \cos(21.04^\circ) = 0.9333$$

$$\tan(\alpha) = \tan(\alpha) \Leftrightarrow \tan(21.04^\circ) = 0.3846$$

$$Q_S = P_0 \cdot \frac{RS}{2} + f_2 = 24.55 \text{ bar} \cdot \frac{1146 \text{ mm}}{2} + 69.5 \text{ N/mm} = 1476 \text{ N/mm}$$

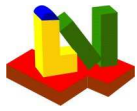
$$A_r S = \frac{k \cdot Q_S \cdot RS}{S_s \cdot E_1} \cdot \left[ 1 - \frac{Del}{a} \right] \cdot \tan(\alpha) = \frac{1 \cdot 1476 \text{ N/mm} \cdot 1146 \text{ mm}}{138 \text{ N/mm}^2 \cdot 1} \cdot \left[ 1 - \frac{11.87^\circ}{21.04^\circ} \right] \cdot 0.3846 = 2055 \text{ mm}^2$$

$$t = \frac{P_0 \cdot RS}{S_s \cdot E_1 - 0.6 \cdot P_0} = \frac{24.55 \text{ bar} \cdot 1146 \text{ mm}}{138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 24.55 \text{ bar}} = 20.61 \text{ mm}$$

$$t_r = \frac{P_0 \cdot \frac{RS}{(\cos(\alpha))}}{S_c \cdot E_2 - 0.6 \cdot P_0} = \frac{24.55 \text{ bar} \cdot \frac{1146 \text{ mm}}{0.9333}}{138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 24.55 \text{ bar}} = 22.08 \text{ mm}$$

$$A_e S = 0.78 \cdot \sqrt{RS \cdot t_s} \cdot \left[ (ts - t) + \frac{(tc - tr)}{\cos(\alpha)} \right] =$$

$$0.78 \cdot \sqrt{1146 \text{ mm} \cdot 25.4 \text{ mm}} \cdot \left[ (25.4 \text{ mm} - 20.61 \text{ mm}) + \frac{(46.04 \text{ mm} - 22.08 \text{ mm})}{0.9333} \right] = 5082 \text{ mm}^2$$

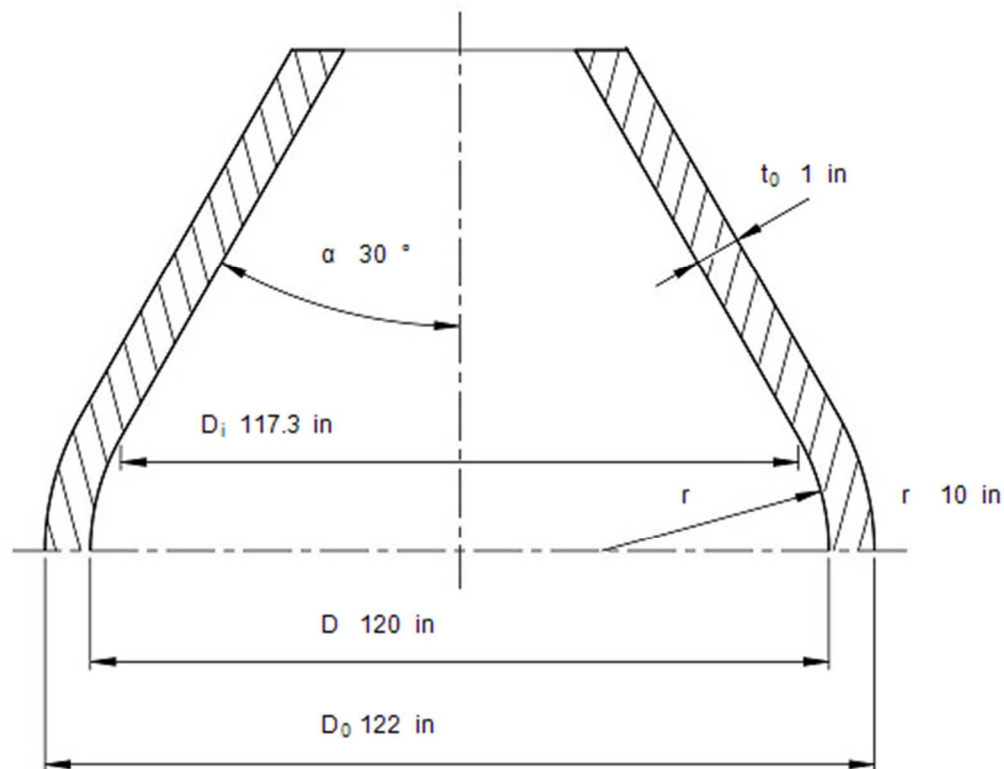


**ASME BPVC VIII-1 2021**  
Example E4.3.7 - E4.3.8 PTB-4-2021

**E4.3.8 - Cone with knuckle under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2021**

**Toriconical sections**

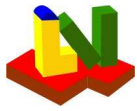
Design pressure	$p_D$	285.4 psi
Hydrostatic head	$D_p$	0 psi
Calculation pressure	$p_0$	<b>285.4</b> psi
Calculation temperature	$T_0$	300 °F
Final wall thickness	$t_e$	1 in
Wall thickness allowance	$c_1$	0 in
Allowance (corrosion)	$c_2$	0 in
Effective thickness without allowances	$t_0$	<b>1</b> in



Outside diameter of cylindrical shell	$D_0$	122 in
Inside diameter of cylindrical shell (= $D_0 - 2t_0$ )	$D$	<b>120</b> in
Semi-apex angle	$\alpha$	30 °
Knuckle radius ( $\geq 0.06 \cdot D_0$ , $\geq 3 \cdot t_0$ )	$r$	10 in
Weld joint efficiency (or Cast Quality Factor)	$E$	1

Material K02700-SA-516-70-Class:-Size:

Allowable stress S **20015** psi



# ASME BPVC VIII-1 2021

## Example E4.3.7 - E4.3.8 PTB-4-2021

### Calculation

Largest inside diameter of cone	$D_i$	117.3 in
Equivalent radius	$L$	67.74 in
Ratio	$L/r$	6.774
Factor	$M$	1.401
Required knuckle thickness	$t$	0.6773 in
Allowable inside pressure of knuckle	$P$	421 psi
Calculation diameter of cone	$D_1$	117.3 in
Required cone thickness at $D_1$	$t_1$	0.9741 in
Allowable inside pressure of cone	$P_1$	292.9 psi
Remark		
Required thickness incl. allowances	$t+C_1+C_2$	$t+$ 0.9741 in
Allowable excess pressure	$\text{Min}(P, P_1)$	$P_m$ 292.9 psi
Allowable excess pressure without hydr. Head		MAWP 292.9 psi

Geometrical conditions  
**valid**

Strength condition  
**Wall thickness acceptable**

### Equations knuckle

$$\cos(\alpha) = \cos(\alpha) \Leftrightarrow \cos(30^\circ) = 0.866$$

$$D_i = D - 2 \cdot r \cdot (1 - \cos(\alpha)) = 3048 \text{ mm} - 2 \cdot 254 \text{ mm} \cdot (1 - 0.866) = 2980 \text{ mm}$$

$$L = \frac{D_i}{2 \cdot \cos(\alpha)} = \frac{2980 \text{ mm}}{2 \cdot 0.866} = 1720 \text{ mm}$$

$$t = \frac{P_0 \cdot L \cdot M}{2 \cdot S \cdot E - 0.2 \cdot P_0} = \frac{19.68 \text{ bar} \cdot 1720 \text{ mm} \cdot 1.401}{2 \cdot 138 \text{ N/mm}^2 \cdot 1 - 0.2 \cdot 19.68 \text{ bar}} = 17.2 \text{ mm}$$

$$P = \frac{2 \cdot S \cdot E \cdot t_0}{L \cdot M + 0.2 \cdot t_0} = \frac{2 \cdot 138 \text{ N/mm}^2 \cdot 1 \cdot 25.4 \text{ mm}}{1720 \text{ mm} \cdot 1.401 + 0.2 \cdot 25.4 \text{ mm}} = 2.903 \text{ MPa}$$

### Equations cone

$$t_1 = \frac{P_0 \cdot D_1}{2 \cdot \cos(\alpha) \cdot (S \cdot E - 0.6 \cdot P_0)} = \frac{19.68 \text{ bar} \cdot 2980 \text{ mm}}{2 \cdot 0.866 \cdot (138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 19.68 \text{ bar})} = 24.74 \text{ mm}$$

$$P = \frac{2 \cdot S \cdot E \cdot t_0 \cdot \cos(\alpha)}{D_1 + 1.2 \cdot t_0 \cdot \cos(\alpha)} = \frac{2 \cdot 138 \text{ N/mm}^2 \cdot 1 \cdot 25.4 \text{ mm} \cdot 0.866}{2980 \text{ mm} + 1.2 \cdot 25.4 \text{ mm} \cdot 0.866} = 2.019 \text{ MPa}$$